

SECTION 17

AMERICAN Engineering Data





Section 17 - Engineering Data

Table of Contents

Energy Savings With Ductile Iron Pipe	17-2
Present Value of Money	17-3
Flow of Water in Ductile Iron Pipe	17-4
Flow of Water in Ductile Iron Pipe (Table 17-1)	17-5 - 17-8
Gravity Flow in Pipe Manning Formula (Fig. 17-1)	17-9
Equivalent Pipe Diameters (Table 17-2).....	17-10
Equivalent Number of Pipelines (Table 17-3).....	17-11
Diameters, Circumferences, Areas and Volumes for Standard Minimum Pressure Classes of Ductile Iron Cement-Lined Pipe (Table 17-4).....	17-12
Weights for Pipeline Design, Weights of Ductile Iron Pipe and Contained Water (Table 17-5)	17-12
Axial Bulkhead or Dead-End Thrust Due to Pressure (Table 17-6).....	17-13
Standard Abbreviations and Acronyms (Table 17-7)	17-14 - 17-20
Taps in Ductile Iron Pipe - Minimum Distances (Table 17-8)	17-21
Standard Pipe Taps - Outside Diameters (Table 17-9)	17-21
Taps in Ductile Iron Pipe - ANSI/ASME B1.20.1 (Table 17-10).....	17-22
Taps in Ductile Iron Pipe - AWWA C800 Standard (Table 17-11)	17-23
Linear Expansion of Ductile Iron Pipe (Table 17-12).....	17-24
Pipe Length Calculations for Offset Connections (Table 17-13).....	17-25
Pipe Length Calculations for Inclined Diagonal Runs.....	17-26
Mathematical Formulas	17-27 - 17-29
AMERICAN's Standard Color Codes.....	17-30
Decimal of an Inch and of a Foot (Table 17-14)	17-31
Equivalents and Conversion Factors - English (Table 17-15)	17-32, 33
Equivalents and Conversion Factors - Metric (Table 17-16)	17-34, 35



Energy Savings With Ductile Iron Pipe

All sizes of standard thickness cement-lined ductile iron pipe in minimum pressure classes and all special thickness classes through Special Thickness Class 52 in 4"-54" sizes have greater than nominal inside diameters. **Head losses in piping are directly related to inside diameters, and energy consumption and accompanying pumping costs are directly related to head losses. Therefore, the use of ductile iron piping having inside diameters greater than nominal can result in significant energy savings over the years. In addition to helping to keep operating costs and utility rates reasonable, this conservation of energy is also helpful to the environment.**

Graphs and tables pertaining to this can be found in "Manual For Computation of Energy Savings With Ductile Iron Pipe" published by the Ductile Iron Pipe Research Association (DIPRA). Formulas for calculating pumping costs are as follows:

$$PC = 1.65 H_L Q \frac{a}{E}$$

where:

PC = Pumping cost in \$/Year/
1000 ft of pipe based on 24
Hr./day Pump Operation

H_L = Hydraulic Gradient or Head Loss in
ft/1000 ft.

Q = Flow in Gallons/Minute (GPM)

a = Unit Cost of Electricity in \$/KWH

E = Total Efficiency of Pump System
(pump, motor, transmission —
always a number less than one)

and: $H_L^* = 1000 \left[\frac{V}{1.318C (r)^{0.63}} \right]^{1.852}$

where:

V = Velocity in ft/sec (fps)

C = Hazen-Williams "C" Factor
(140 for ductile iron pipe)**

r = Hydraulic Radius in ft (inside diameter
in ft ÷ 4 for pipe flowing full)

and:

$$V^* = \frac{Q}{448.8A}$$

where:

Q = Flow in Gallons/Minute (GPM)

A = Cross sectional area of pipe in ft²

and:

$$A^{***} = \frac{\pi d^2}{4}$$

where:

d*** = Inside diameter in ft

*See Table No. 17-1 for values of H_L and V for standard minimum pressure class cement-lined ductile iron pipe.

**The Hazen-Williams flow coefficient C=140 has been used for cement-lined cast iron and ductile iron pipe for many years. The quality of recent, high-performance cement linings for ductile iron pipe, smooth proprietary linings, and the availability of even larger pipe sizes may justify the use of significantly higher values for C. This may be particularly applicable to the intermediate and larger pipe sizes in clean water service. (See pg 17-4.)

***See Table No. 17-4 for values of A and d for standard minimum pressure class cement-lined ductile iron pipe.



Present Value of Money

The present value of money compounded annually at a given investment rate and an assumed inflation rate over a given period can be calculated by the following formulas:

$$PV = \frac{(1 + i)^n - 1}{i(1 + i)^n}$$

where:

PV = Present Value per \$1

i = Effective interest rate (%/100)

n = Number of compounding periods (years)

$$\text{and } i = \frac{j - i'}{1 + i'}$$

where:

j = Investment rate (%/100)

i' = Inflation rate (%/100)

Example:

Given:

50,000' 36" Pressure Class 150 pipe
Design flow — 14,000 GPM
Unit Power Cost — \$.05 KWH
Pump Operation — 24 Hrs/Day
Pump System Efficiency — 70%
Hazen-Williams "C" Factor — 140

Using the Pumping Cost formula on page 17-2, Pressure Class 150, cement-lined, ductile iron pipe will save approximately \$18,075/year in Pumping Costs over a substitute material pipe with a nominal inside diameter.

The Present Value of the \$18,075 annual savings, over a 50-year time period, and assuming an investment rate of 8% and an inflation rate of 4%, is \$397,650 (\$18,075 x 22).



Flow of Water In Ductile Iron Pipe

The carrying capacity of a given pipeline is limited by its internal resistance to the flow of water. This resistance to flow causes a loss of head or drop in pressure as the water moves through the line. The amount of head loss depends on (1) the velocity of the water, (2) the roughness of the interior surface of the pipe, (3) the internal diameter, and (4) the length of the line. These factors have been related in the widely used Hazen-Williams formula for computing head losses, pipe sizes and carrying capacities in distribution lines. This formula is as follows:

$$V = C \times r^{0.63} \times S^{0.54} \times 0.001^{-0.04}$$

in which

V = velocity of water through the pipe in feet per second

C = factor depending on the roughness of the interior surface

r = hydraulic radius which is $\frac{1}{4}$ the internal diameter (for pipe flowing full), in feet

S = hydraulic slope or head loss in feet per foot of pipe

The factor C is well known as the Hazen-Williams "C" or flow coefficient "C," and its value must be estimated in flow calculations.

Numerous tests have shown that cement-lined pipe installed many years ago maintains a nearly constant "C" of 140 even in tuberculating waters. The quality of more recent, high-performance AMERICAN cement linings and the availability of even larger pipe sizes may justify the use of significantly higher values for C, particularly in intermediate and larger pipe sizes. Multiple hydraulic flow tests of 42" AMERICAN "high performance" cement-mortar-lined pipeline (with standard, high-speed spun cement lining for that pipe size) conducted in 2001 at the flow facility at Utah State University resulted in an experimentally determined average "C" value slightly greater than 152, with flow velocities in the 3-13 fps (1-4 m/sec) range. A range of Mannings "n" values of 0.0089-0.010 for clean water was also determined for this same high-performance cement lining in this same 42" Utah State testing program. The Utah State data also showed the resulting Darcy friction factors "f" for the high-performance cement lining, when plotted vs. Reynolds number, very closely approximate the "smooth pipe curve" as shown on the Moody diagram.



Flow of Water in Ductile Iron Pipe

Hazen-Williams C=140*

Loss of Head shown is per 1,000 feet of pipeline. Table is based on 4"-64" AWWA C104 single-thickness cement-lined ductile iron pipe. Nominal thickness for minimum standard pressure classes noted.

Table No. 17-1

Flow in Gallons per 24 Hours	4" Pipe, Class 350		6" Pipe, Class 350		8" Pipe, Class 350		10" Pipe, Class 350		12" Pipe, Class 350	
	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)
20,000	.33	.14								
30,000	.49	.29								
40,000	.65	.50								
50,000	.81	.75	.36	.10						
60,000	.98	1.05	.43	.14						
70,000	1.14	1.40	.50	.19						
80,000	1.30	1.79	.58	.25						
90,000	1.46	2.23	.65	.31						
100,000	1.63	2.71	.72	.37						
110,000	1.79	3.23	.79	.44						
120,000	1.95	3.79	.86	.52						
140,000	2.28	5.05	1.01	.69						
160,000	2.60	6.46	1.15	.89						
180,000	2.93	8.04	1.30	1.11						
200,000	3.25	9.77	1.44	1.34	.80	.32				
220,000	3.58	11.66	1.58	1.60	.88	.38				
240,000	3.91	13.70	1.73	1.88	.96	.45				
260,000	4.23	15.88	1.87	2.18	1.04	.52				
280,000	4.56	18.22	2.02	2.50	1.12	.60				
300,000	4.88	20.70	2.16	2.85	1.20	.68	.78	.24	.54	.10
320,000	5.21	23.33	2.31	3.21	1.28	.76	.83	.27	.58	.11
340,000	5.53	26.11	2.45	3.59	1.36	.85	.88	.30	.62	.12
360,000	5.86	29.02	2.59	3.99	1.44	.95	.93	.33	.65	.14
380,000	6.18	32.08	2.74	4.41	1.52	1.05	.99	.37	.69	.15
400,000	6.51	35.27	2.88	4.85	1.60	1.15	1.04	.40	.72	.17
450,000	7.32	43.87	3.24	6.03	1.80	1.44	1.17	.50	.82	.21
500,000	8.14	53.32	3.60	7.33	2.00	1.75	1.30	.61	.91	.25
550,000	8.95	63.61	3.96	8.74	2.20	2.08	1.43	.73	1.00	.30
600,000	9.76	74.74	4.32	10.27	2.40	2.45	1.56	.85	1.09	.36
650,000	10.58	86.68	4.68	11.91	2.60	2.84	1.69	.99	1.18	.41
700,000			5.04	13.67	2.80	3.25	1.82	1.14	1.27	.47
750,000			5.40	15.53	3.00	3.70	1.95	1.29	1.36	.54
800,000			5.76	17.50	3.20	4.17	2.08	1.46	1.45	.61
900,000			6.48	21.77	3.60	5.18	2.34	1.81	1.63	.75
1,000,000			7.20	26.46	4.00	6.30	2.60	2.20	1.81	.92
1,200,000			8.65	37.08	4.80	8.83	3.11	3.09	2.17	1.29
1,400,000			10.09	49.33	5.60	11.75	3.63	4.10	2.54	1.71
1,600,000					6.39	15.04	4.15	5.26	2.90	2.19
1,800,000					7.19	18.71	4.67	6.54	3.26	2.72
2,000,000					7.99	22.74	5.19	7.95	3.62	3.31
2,200,000					8.79	27.13	5.71	9.48	3.98	3.95
2,400,000					9.59	31.87	6.23	11.14	4.35	4.64
2,600,000					10.39	36.96	6.75	12.92	4.71	5.38
2,800,000							7.27	14.82	5.07	6.17
3,000,000							7.79	16.84	5.43	7.01
3,500,000							9.08	22.40	6.34	9.33
4,000,000							10.38	28.68	7.24	11.95
4,500,000									8.15	14.86
5,000,000									9.06	18.06
5,500,000									9.96	21.54
6,000,000									10.87	25.31

*The Hazen-Williams flow coefficient C=140 has been used for cement-lined cast iron and ductile iron pipe for many years. The quality of recent high-performance cement lining for ductile iron pipe and the availability of even larger pipe sizes may justify the use of significantly higher values for C, particularly in the intermediate and larger pipe sizes. (See pg 17-4.)

The design of systems outside common water velocities, i.e. 2-5 fps, may involve special design considerations (for example, the generation of substantial surge pressures as a result of valve closure or other water column effects, sedimentation at extremely low velocities, etc.).



Flow of Water in Ductile Iron Pipe

Hazen-Williams C=140*

Loss of Head shown is per 1,000 feet of pipeline. Table is based on 4"-64" AWWA C104 single-thickness cement-lined ductile iron pipe. Nominal thickness for minimum standard pressure classes noted.

Table No. 17-1—Continued

Flow in Gallons per 24 Hours	14" Pipe, Class 250		16" Pipe, Class 250		18" Pipe, Class 250		20" Pipe, Class 250	
	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)
300,000	.40	.05						
400,000	.54	.08	.41	.04	.32	.02		
500,000	.67	.12	.51	.06	.41	.04		
600,000	.80	.17	.62	.09	.49	.05	.40	.03
700,000	.94	.23	.72	.12	.57	.07	.46	.04
800,000	1.07	.29	.82	.15	.65	.09	.53	.05
900,000	1.21	.36	.93	.19	.73	.11	.59	.06
1,000,000	1.34	.44	1.03	.23	.81	.13	.66	.08
1,100,000	1.47	.52	1.13	.28	.89	.15	.72	.09
1,200,000	1.61	.62	1.23	.32	.97	.18	.79	.11
1,300,000	1.74	.71	1.34	.38	1.06	.21	.86	.13
1,400,000	1.88	.82	1.44	.43	1.14	.24	.92	.15
1,500,000	2.01	.93	1.54	.49	1.22	.28	.99	.17
1,600,000	2.14	1.05	1.64	.55	1.30	.31	1.05	.19
1,700,000	2.28	1.17	1.75	.62	1.38	.35	1.12	.21
1,800,000	2.41	1.31	1.85	.69	1.46	.39	1.19	.23
1,900,000	2.55	1.44	1.95	.76	1.54	.43	1.25	.26
2,000,000	2.68	1.59	2.06	.83	1.62	.47	1.32	.28
2,200,000	2.95	1.89	2.26	.99	1.79	.56	1.45	.34
2,400,000	3.21	2.23	2.47	1.17	1.95	.66	1.58	.40
2,500,000	3.35	2.40	2.57	1.26	2.03	.71	1.65	.43
2,600,000	3.48	2.58	2.67	1.35	2.11	.76	1.71	.46
2,800,000	3.75	2.96	2.88	1.55	2.27	.87	1.84	.53
3,000,000	4.02	3.36	3.08	1.76	2.44	.99	1.98	.60
3,500,000	4.69	4.47	3.60	2.35	2.84	1.32	2.31	.79
4,000,000	5.36	5.73	4.11	3.01	3.25	1.69	2.63	1.02
4,500,000	6.03	7.13	4.63	3.74	3.65	2.11	2.96	1.27
5,000,000	6.70	8.66	5.14	4.55	4.06	2.56	3.29	1.54
5,500,000	7.37	10.33	5.65	5.42	4.47	3.05	3.62	1.83
6,000,000	8.04	12.14	6.17	6.37	4.87	3.59	3.95	2.16
6,500,000	8.71	14.08	6.68	7.39	5.28	4.16	4.28	2.50
7,000,000	9.38	16.15	7.19	8.48	5.68	4.77	4.61	2.87
7,500,000	10.05	18.35	7.71	9.63	6.09	5.42	4.94	3.26
8,000,000			8.22	10.85	6.49	6.11	5.27	3.67
8,500,000			8.74	12.14	6.90	6.84	5.60	4.11
9,000,000			9.25	13.50	7.31	7.60	5.93	4.57
9,500,000			9.76	14.92	7.71	8.40	6.26	5.05
10,000,000			10.28	16.41	8.12	9.24	6.59	5.55
11,000,000					8.93	11.02	7.25	6.62
12,000,000					9.74	12.95	7.90	7.78
13,000,000					10.55	15.01	8.56	9.02
14,000,000							9.22	10.35
15,000,000							9.88	11.76
16,000,000							10.54	13.25

*The Hazen-Williams flow coefficient C=140 has been used for cement-lined cast iron and ductile iron pipe for many years. The quality of recent high-performance cement lining for ductile iron pipe and the availability of even larger pipe sizes may justify the use of significantly higher values for C, particularly in the intermediate and larger pipe sizes. (See pg 17-4.)

The design of systems outside common water velocities, i.e. 2-5 fps, may involve special design considerations (for example, the generation of substantial surge pressures as a result of valve closure or other water column effects, sedimentation at extremely low velocities, etc.).



Flow of Water in Ductile Iron Pipe

Hazen-Williams C=140*

Loss of Head shown is per 1,000 feet of pipeline. Table is based on 4"-64" AWWA C104 single-thickness cement-lined ductile iron pipe. Nominal thickness for minimum standard pressure classes noted.

Table No. 17-1—Continued

Flow in Gallons per 24 Hours	24" Pipe, Class 200		30" Pipe, Class 150		36" Pipe, Class 150		42" Pipe, Class 150	
	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)
1,000,000	.46	.03						
1,200,000	.55	.04						
1,400,000	.64	.06						
1,500,000	.68	.07	.44	.02	.31	.01		
1,600,000	.73	.08	.47	.03	.33	.01		
1,800,000	.82	.09	.53	.03	.37	.01		
2,000,000	.91	.11	.59	.04	.41	.02		
2,500,000	1.14	.17	.73	.06	.51	.02		
3,000,000	1.37	.24	.88	.08	.61	.03	.45	.02
3,500,000	1.59	.32	1.03	.11	.71	.05	.53	.02
4,000,000	1.82	.41	1.18	.14	.82	.06	.60	.03
4,500,000	2.05	.52	1.32	.18	.92	.07	.68	.03
5,000,000	2.28	.63	1.47	.22	1.02	.09	.75	.04
5,500,000	2.51	.75	1.62	.26	1.12	.11	.83	.05
6,000,000	2.73	.88	1.76	.30	1.22	.12	.90	.06
6,500,000	2.96	1.02	1.91	.35	1.33	.14	.98	.07
7,000,000	3.19	1.17	2.06	.40	1.43	.17	1.05	.08
7,500,000	3.42	1.33	2.20	.46	1.53	.19	1.13	.09
8,000,000	3.64	1.50	2.35	.51	1.63	.21	1.20	.10
8,500,000	3.87	1.67	2.50	.58	1.73	.24	1.28	.11
9,000,000	4.10	1.86	2.64	.64	1.84	.26	1.35	.13
9,500,000	4.33	2.06	2.79	.71	1.94	.29	1.43	.14
10,000,000	4.56	2.26	2.94	.78	2.04	.32	1.50	.15
11,000,000	5.01	2.70	3.23	.93	2.24	.38	1.65	.18
12,000,000	5.47	3.17	3.53	1.09	2.45	.45	1.80	.21
13,000,000	5.92	3.68	3.82	1.26	2.65	.52	1.96	.25
14,000,000	6.38	4.22	4.11	1.45	2.86	.60	2.11	.28
15,000,000	6.83	4.79	4.41	1.65	3.06	.68	2.26	.32
16,000,000	7.29	5.49	4.70	1.86	3.26	.76	2.41	.36
18,000,000	8.20	6.72	5.29	2.31	3.67	.95	2.71	.45
20,000,000	9.11	8.17	5.88	2.81	4.08	1.15	3.01	.55
22,000,000	10.02	9.74	6.46	3.35	4.49	1.38	3.31	.66
24,000,000			7.05	3.93	4.90	1.62	3.61	.77
26,000,000			7.64	4.56	5.30	1.88	3.91	.89
28,000,000			8.23	5.23	5.71	2.15	4.21	1.02
30,000,000			8.82	5.95	6.12	2.45	4.51	1.16
32,000,000			9.40	6.70	6.53	2.76	4.81	1.31
34,000,000			9.99	7.50	6.94	3.08	5.11	1.47
36,000,000			10.58	8.34	7.34	3.43	5.41	1.63
38,000,000					7.75	3.79	5.72	1.80
40,000,000					8.16	4.17	6.02	1.92
45,000,000					9.18	5.18	6.77	2.47
50,000,000					10.20	6.30	7.52	3.00
55,000,000							8.27	3.58
60,000,000							9.02	4.20
65,000,000							9.78	4.87
70,000,000							10.53	5.59

*The Hazen-Williams flow coefficient C=140 has been used for cement-lined cast iron and ductile iron pipe for many years. The quality of recent high-performance cement lining for ductile iron pipe and the availability of even larger pipe sizes may justify the use of significantly higher values for C, particularly in the intermediate and larger pipe sizes. (See pg 17-4.)

The design of systems outside common water velocities, i.e. 2-5 fps, may involve special design considerations (for example, the generation of substantial surge pressures as a result of valve closure or other water column effects, sedimentation at extremely low velocities, etc.).



Flow of Water in Ductile Iron Pipe

Hazen-Williams C=140*

Loss of Head shown is per 1,000 feet of pipeline. Table is based on 4"-64" AWWA C104 single-thickness cement-lined ductile iron pipe. Nominal thickness for minimum standard pressure classes noted.

Table No. 17-1—Continued

Flow in Gallons per 24 Hours	48" Pipe, Class 150		54" Pipe, Class 150		60" Pipe, Class 150		64" Pipe, Class 150	
	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)	Vel. in ft. per sec.	Loss of Head (ft)
3,000,000	.35	.01						
4,000,000	.46	.01						
5,000,000	.58	.02						
6,000,000	.69	.03	.54	.02	.47	.01	.41	.01
7,000,000	.81	.04	.63	.02	.55	.02	.48	.01
8,000,000	.92	.05	.72	.03	.62	.02	.55	.01
9,000,000	1.04	.07	.81	.04	.70	.03	.62	.02
10,000,000	1.15	.08	.90	.04	.78	.03	.69	.02
12,000,000	1.38	.11	1.07	.06	.94	.04	.82	.03
14,000,000	1.61	.15	1.25	.08	1.09	.06	.96	.04
16,000,000	1.84	.19	1.43	.10	1.25	.07	1.10	.05
18,000,000	2.07	.24	1.61	.13	1.41	.09	1.24	.07
20,000,000	2.30	.29	1.79	.16	1.56	.11	1.37	.08
22,000,000	2.53	.34	1.97	.19	1.72	.13	1.51	.10
24,000,000	2.76	.40	2.15	.22	1.87	.16	1.65	.11
26,000,000	2.99	.47	2.33	.25	2.03	.18	1.78	.13
28,000,000	3.22	.53	2.51	.29	2.19	.21	1.92	.15
30,000,000	3.46	.61	2.69	.33	2.34	.24	2.06	.17
32,000,000	3.69	.68	2.86	.37	2.50	.27	2.20	.19
34,000,000	3.92	.77	3.04	.41	2.65	.30	2.33	.22
36,000,000	4.15	.85	3.22	.46	2.81	.33	2.47	.24
38,000,000	4.38	.94	3.40	.51	2.97	.37	2.61	.27
40,000,000	4.61	1.04	3.58	.56	3.12	.40	2.74	.29
45,000,000	5.18	1.29	4.03	.70	3.51	.50	3.09	.36
50,000,000	5.76	1.57	4.48	.85	3.90	.61	3.43	.44
55,000,000	6.33	1.87	4.92	1.01	4.29	.72	3.77	.53
60,000,000	6.91	2.19	5.37	1.19	4.68	.85	4.12	.62
65,000,000	7.49	2.54	5.82	1.38	5.07	.99	4.46	.72
70,000,000	8.06	2.92	6.27	1.58	5.46	1.13	4.80	.83
75,000,000	8.64	3.32	6.71	1.80	5.86	1.29	5.15	.94
80,000,000	9.21	3.74	7.16	2.02	6.25	1.45	5.49	1.06
85,000,000	9.79	4.18	7.61	2.26	6.64	1.62	5.83	1.18
90,000,000	10.37	4.65	8.06	2.52	7.03	1.80	6.18	1.32
95,000,000			8.51	2.78	7.42	1.99	6.52	1.46
100,000,000			8.95	3.06	7.81	2.19	6.86	1.60
105,000,000			9.40	3.35	8.20	2.40	7.20	1.75
110,000,000			9.85	3.65	8.59	2.61	7.55	1.91
115,000,000			10.30	3.96	8.98	2.84	7.89	2.07
120,000,000					9.37	3.07	8.23	2.24
125,000,000					9.76	3.31	8.58	2.42
130,000,000					10.15	3.56	8.92	2.60
135,000,000							9.26	2.79
140,000,000							9.61	2.98
145,000,000							9.95	3.18
150,000,000							10.29	3.39

*The Hazen-Williams flow coefficient C=140 has been used for cement-lined cast iron and ductile iron pipe for many years. The quality of recent high-performance cement lining for ductile iron pipe and the availability of even larger pipe sizes may justify the use of significantly higher values for C, particularly in the intermediate and larger pipe sizes. (See pg 17-4.)

The design of systems outside common water velocities, i.e. 2-5 fps, may involve special design considerations (for example, the generation of substantial surge pressures as a result of valve closure or other water column effects, sedimentation at extremely low velocities, etc.).



Gravity Flow in Pipe Manning Formula

FORMULAS:

— for pipe flowing full —

$$Q = \frac{0.276}{n} \times D^{\frac{8}{3}} \times S^{\frac{1}{2}}$$

$$V = \frac{0.1125}{n} \times D^{\frac{2}{3}} \times S^{\frac{1}{2}}$$

Q = Flow in gallons per minute

D = Pipe diameter in inches

S = Slope in feet per foot

V = Velocity in feet per second

Protecto 401-lined Ductile Iron pipe has an approximate "n" value of n = 0.011*.

Example:

For a 20" Ductile Iron Protecto 401-lined pipe, Pressure Class 250, with a slope of 4.5 feet per 1000 feet, find the flow "Q" and the velocity "V" when the pipe is flowing full. The roughness coefficient "n" is 0.011.

$$Q = \frac{.276 \times (20.86)^{\frac{8}{3}} \times (0.0045)^{\frac{1}{2}}}{0.011}$$

$$Q = 5550 \text{ gals./min.}$$

$$V = \frac{.1125 \times (20.86)^{\frac{2}{3}} \times (0.0045)^{\frac{1}{2}}}{0.011}$$

$$V = 5.20 \text{ feet/sec.}$$

Nominal Size	Nominal I.D.	Thickness Class
64"	64.47"	Class 150
60"	60.45"	Class 150
56"	56.46"	Class 150
54"	54.47"	Class 150
48"	49.80"	Class 150
42"	43.60"	Class 150
36"	37.46"	Class 150
30"	31.24"	Class 150
24"	25.06"	Class 200
20"	20.86"	Class 250
18"	18.80"	Class 250
16"	16.72"	Class 250
14"	14.66"	Class 250
12"	12.56"	Class 350
10"	10.50"	Class 350
8"	8.47"	Class 350
6"	6.32"	Class 350

Note: *Per Protecto 401 Web site at:
<http://www.protecto401.com/n-cfactor.htm>

Ductile Nominal Minimum Pressure Class
Thickness Protecto 401-Lined Pipe

Fig. 17-1



Equivalent Pipe Diameters

Table No. 17-2

Nominal Dia. of Pipe "A"	Nominal Dia. of Pipe "B"								
	2	3	4	6	8	10	12	14	16
2	2.6	3.36	4.23	6.12	8.08				
3		3.90	4.63	6.35	8.23	10.16			
4			5.21	6.71	8.47	10.33	12.25		
6				7.81	9.26	10.92	12.70	14.56	16.45
8					10.41	11.83	13.43	15.14	16.94
10						13.02	14.41	15.97	17.63
12							15.62	17.00	18.52
14								18.22	19.59
16									20.82
18									
20									
24									
30									
36									
42									
48									
54									
60									
64									

Table No. 17-2—Continued

Nominal Dia. of Pipe "A"	Nominal Dia. of Pipe "B"									
	18	20	24	30	36	42	48	54	60	64
2										
3										
4										
6										
8	18.78	20.66	24.50	30.35	36.26					
10	19.37	21.17	24.89	30.62	36.47	42.36				
12	20.14	21.84	25.41	31.00	36.75	42.59	48.47	54.39	60.33	64.30
14	21.09	22.68	26.06	31.48	37.11	42.87	48.71	54.58	60.49	64.44
16	22.19	23.66	26.86	32.07	37.57	43.23	49.00	54.83	60.70	64.63
18	23.43	24.79	27.78	32.76	38.11	43.67	49.35	55.12	60.95	64.86
20		26.03	28.83	33.57	38.74	44.18	49.77	55.47	61.25	65.13
24			31.24	35.49	40.29	45.43	50.81	56.35	61.99	65.80
30				39.05	43.24	47.90	52.89	58.12	63.52	67.19
36					46.86	51.00	55.56	60.43	65.53	69.03
42						54.67	58.78	63.26	68.03	71.34
48							62.47	66.57	70.98	74.08
54								70.28	74.36	77.24
60									78.09	80.76
64										83.30

All figures are in inches.

Example: In parallel, a 6-inch and an 8-inch main are only equivalent to one pipe 9.26 inches in diameter.

The equivalent size pipe required to equal, at the same head, the carrying capacity of two smaller pipes is tabulated in this table. This evaluation is based on the comparative nominal diameters raised to the 2.63 power.



Equivalent Number of Pipelines

Tabulated below are the approximate number of pipes of a given size equal in carrying capacity to one pipe of a larger size. At the same velocity of flow the volume delivered by two pipes of different sizes is proportional to the squares of their diameters. With the same head, however, the velocity is less in the smaller pipe, and the volume delivered varies approximately as the diameters to the 2.63 power. This table is calculated on this basis using nominal diameters. The figure opposite the intersection of any two sizes is the number of the smaller-sized pipes required to equal one of the larger; thus one 4-inch pipe equals 6.2 two-inch pipes.

Table No. 17-3

Dia. in.	1/2	3/4	1	2	3	4	5	6	7	8	10	12
2	38.3	13.2	6.2	1.0								
3	111.3	38.3	18.0	2.9	1.0							
4	237.2	81.7	38.3	6.2	2.1	1.0						
6	689.0	237.2	111.3	18.0	6.2	2.9	1.6	1.0				
8		505.5	237.2	38.3	13.2	6.2	3.4	2.1	1.4	1.0		
10		909.1	426.6	68.9	23.7	11.1	6.2	3.8	2.6	1.8	1.0	
12			689.0	111.3	38.3	18.0	10.0	6.2	4.1	2.9	1.6	1.0
14				167.0	57.5	27.0	15.0	9.3	6.2	4.3	2.4	1.5
16				237.2	81.7	38.3	21.3	13.2	8.8	6.2	3.4	2.1
18				323.3	111.3	52.2	29.0	18.0	12.0	8.4	4.7	2.9
20				426.6	146.9	68.9	38.3	23.7	15.8	11.1	6.2	3.8
24				689.0	237.2	111.3	61.9	38.3	25.5	18.0	10.0	6.2
30						200.2	111.3	68.9	45.9	32.3	18.0	11.1
36								111.3	74.2	52.2	29.1	18.0
42										78.3	43.6	27.0
48											61.9	38.3
54												52.2
60												68.9
64												81.7

Table No. 17-3—Continued

Dia. in.	14	16	18	20	24	30	36	42	48	54	60	64
2												
3												
4												
6												
8												
10												
12												
14	1.0											
16	1.4	1.0										
18	1.9	1.4	1.0									
20	2.6	1.8	1.3	1.0								
24	4.1	2.9	2.1	1.6	1.0							
30	7.4	5.2	3.8	2.9	1.8	1.0						
36	12.0	8.4	6.2	4.7	2.9	1.6	1.0					
42	18.0	12.7	9.3	7.0	4.4	2.4	1.5	1.0				
48	25.5	18.0	13.2	10.0	6.2	3.4	2.1	1.4	1.0			
54	34.8	24.5	18.0	13.6	8.4	4.7	2.9	1.9	1.4	1.0		
60	45.9	32.3	23.7	18.0	11.1	6.2	3.8	2.6	1.8	1.3	1.0	
64	54.4	38.3	28.1	21.3	13.2	7.3	4.5	3.0	2.1	1.6	1.2	1.0



Diameters, Circumferences, Areas and Volumes for Standard Minimum Pressure Classes of Ductile Iron Single-Thickness Cement-Lined Pipe

Table No. 17-4

Nominal Size in.	Outside Diameter in.	Inside Diameter in.	Cross Sectional Areas		O.D. Circum- ference in.	I.D. Circum- ference in.	Approx. Outside Surface Area* sq. ft.	Volume Gallons per linear ft.	Volume† Gallons per 20' nom. length
			Based on O.D. sq. in.	Based on I.D. sq. in.					
4	4.80	4.17	18.10	13.66	15.08	13.10	26	0.71	14.19
6	6.90	6.27	37.39	30.88	21.68	19.70	37	1.60	32.08
8	9.05	8.42	64.33	55.68	28.43	26.45	48	2.89	57.85
10	11.10	10.45	96.77	85.77	34.87	32.83	59	4.46	89.11
12	13.20	12.51	136.85	122.92	41.47	39.30	70	6.39	127.71
14	15.30	14.55	183.85	166.27	48.07	45.71	82	8.64	172.75
16	17.40	16.61	237.79	216.69	54.66	52.18	93	11.26	225.13
18	19.50	18.69	298.65	274.35	61.26	58.72	105	14.25	285.04
20	21.60	20.75	366.44	338.16	67.86	65.19	116	17.57	351.34
24	25.80	24.95	522.79	488.91	81.05	78.38	138	25.40	507.96
30	32.00	31.07	804.25	758.18	100.83	97.61	172	39.39	787.72
36	38.30	37.29	1152.10	1092.13	120.32	117.15	206	56.73	1134.68
42	44.50	43.43	1555.29	1481.39	139.80	136.44	240	76.96	1539.11
48	50.80	49.63	2026.83	1934.55	159.59	155.92	275	100.50	2009.92
54	57.56	56.29	2602.15	2488.59	180.83	176.84	312	129.28	2585.55
60	61.61	60.28	2981.22	2853.89	193.55	189.37	334	148.25	2965.08
64	65.67	64.30	3387.08	3247.23	206.31	202.00	357	168.69	3373.75

*This value is roughly calculated as the outside surface area of a cylinder with an O.D. equal to the standard barrel O.D. of the pipe and length equal to the overall length of a "full-length" Fastite® pipe. Obviously, cut or trim pipe would have less surface area. Also, if one wanted to include the flare effects on area as a result of the sloping bell contour or the area of the radial bell face, this would increase the areas very slightly from the tabular values shown.

†This value may also be helpful in a rough determination of the "buoyancy" of a ductile iron pipe full of air. (Buoyancy force is roughly equivalent to weight of fresh or salt water, etc., displaced.) See also Section 9 for buoyancy of 4"-36" Flex-Ring pc 350 pipe.

Weights for Pipeline Design, Weights of Ductile Iron Pipe and Contained Water

Table No. 17-5

Size in.	Weight—Pounds per Foot			Size in.	Weight—Pounds per Foot		
	Pipe	Water	Total		Pipe	Water	Total
4	12	6	18	24	93	212	305
6	18	13	31	30	123	329	452
8	24	24	48	36	163	473	636
10	30	37	67	42	206	642	848
12	39	53	92	48	261	838	1099
14	47	72	119	54	325	1078	1403
16	57	94	151	60	371	1237	1608
18	66	119	185	64	410	1407	1817
20	78	147	225				

These weights are based on minimum pressure classes Ductile Iron Fastite® pipe with minimum thickness standard cement lining as specified in AWWA C104 and on weight of water of 62.4 pounds per cubic foot. All weights are as "in air" at sea level with no buoyancy effects considered. The inside diameters are given in Table No. 17-4.

Pounds of water per foot = $0.3403 \times (\text{I.D. in inches})^2$.

Ductile Iron = 0.255 pounds per cubic inch.



Axial Bulkhead or Dead-End Thrust Due to Pressure*

Table No. 17-6

Nominal Pipe Size in.	Pipe O.D. For Pressure Area Calc.	Pressure (psi)														
		0	15	50	100	150	200	225	250	300	350	375	400	450	500	525
4	4.80	0	300	900	1,800	2,700	3,600	4,100	4,500	5,400	6,300	6,800	7,200	8,100	9,000	9,500
6	6.90	0	600	1,900	3,700	5,600	7,500	8,400	9,300	11,200	13,100	14,000	15,000	16,800	18,700	19,600
8	9.05	0	1,000	3,200	6,400	9,600	12,900	14,500	16,100	19,300	22,500	24,100	25,700	28,900	32,200	33,800
10	11.10	0	1,500	4,800	9,700	14,500	19,400	21,800	24,200	29,000	33,900	36,300	38,700	43,500	48,400	50,800
12	13.20	0	2,100	6,800	13,700	20,500	27,400	30,800	34,200	41,100	47,900	51,300	54,700	61,600	68,400	71,800
14	15.30	0	2,800	9,200	18,400	27,600	36,800	41,400	46,000	55,200	64,300	68,900	73,500	82,700	91,900	96,500
16	17.40	0	3,600	11,900	23,800	35,700	47,600	53,500	59,400	71,300	83,200	89,200	95,100	107,000	118,900	124,800
18	19.50	0	4,500	14,900	29,900	44,800	59,700	67,200	74,700	89,600	104,500	112,000	119,500	134,400	149,300	156,800
20	21.60	0	5,500	18,300	36,600	55,000	73,300	82,400	91,600	109,900	128,200	137,400	146,600	164,900	183,200	192,400
24	25.80	0	7,800	26,100	52,300	78,400	104,600	117,600	130,700	156,800	183,000	196,000	209,100	235,200	261,400	274,500
30	32.00	0	12,100	40,200	80,400	120,600	160,800	181,000	201,100	241,300	281,500	301,600	321,700	361,900	402,100	422,200
36	38.30	0	17,300	57,600	115,200	172,800	230,400	259,200	288,000	345,600	403,200	432,000	460,800	518,400	576,000	604,800
42	44.50	0	23,300	77,800	155,500	233,300	311,000	349,900	388,800	466,600	544,300	583,200	622,100	699,900	777,600	816,500
48	50.80	0	30,400	101,300	202,700	304,000	405,400	456,000	506,700	608,000	709,400	760,000	810,700	912,000	1,013,400	1,064,100
54	57.56	0	39,000	130,100	260,200	390,300	520,400	585,500	650,500	780,600	910,700	975,800	1,040,800	1,170,900	1,301,000	1,366,100
60	61.61	0	44,700	149,100	298,100	447,200	596,200	670,800	745,300	894,300	1,043,400	1,117,900	1,192,400	1,341,500	1,490,600	1,565,100
64	65.67	0	50,800	169,300	338,700	508,000	677,400	762,100	846,700	1,016,100	1,185,400	1,270,100	1,354,800	1,524,100	1,693,500	1,778,200

*This value represents the force in pounds that will be exerted on a test bulkhead (plug, cap, blind flange, etc.), a closed valve (disk, gate, ball, wedge, cone, etc.), or the back of a full-opening tee by the aggregate effect of internal pressure acting over the pressure area (PA). The tabular numbers do not include any provision for safety factors nor momentum effects such as surge pressures in line filling or operation over the pressure values shown. The manual "Thrust Restraint Design for Ductile Iron Pipe" contains a more comprehensive discussion on the subjects of thrust forces and thrust restraint.



Standard Abbreviations and Acronyms

Listed below are many of the standard abbreviations AMERICAN uses on quotations, line drawings, sales order contracts, shipping lists and invoices.

Table No. 17-7

Product or Term Identification or Description	Standard Abbreviation
A	
Accessories.....	ACC
Adjustment.....	ADJ
Air Test.....	A/T
Allocated.....	ALLOC
Alloy.....	ALLOY
Alloy Steel Bolts THHN.....	BOLT THHN
All Mechanical Joint Tee.....	MJ TEE
American Cast Iron Pipe Company.....	ACIPCO
American-Darling Valve.....	ADV
American Ductile Iron Pipe Division.....	ADIP
American Fastite Pipe, Ductile Iron.....	FST PIPE
American Flex-Lok Ball & Socket Pipe, Ductile Iron.....	FLEXLOK PIPE (or FLXLK)
American Flow Control Division.....	AFC
Armor TIP (MJ).....	MJARMT
Asphaltic Coated Inside & Outside.....	CTD
Asphaltic Coated Outside & Cement Lined Inside.....	C/L
Asphaltic Coated Outside & Double Thickness Cement Lined Inside.....	DBL C/L
Assembled.....	ASSEMB
Assigned Not Shipped.....	ANS
As Soon As Possible.....	ASAP
Atlanta District Office.....	ATL
B	
Back Face.....	BF
Ball End.....	BALL
Base Faced & Drilled.....	BA F&D
Base Drilled.....	BA DR
Base (Tee or Bend).....	BASE
Bell Gland.....	BG
Bell Joint Clamp.....	BELL JT CLAMP
Bell Joint Clamp Gasket.....	BELL JT CLAMP GSK
Bell & Spigot.....	BS (Designate GA)
Bend.....	BEND
Beveled.....	BEV
Bill of Lading.....	B/L
Bill of Material.....	BOM
Birmingham District Office.....	BHM
Blind.....	BLD
Blind Flange.....	BLD FLG
Blind Flange Tap 1" (NPT or IPS) at Invert.....	BLD FLG TAP 1 INV
Blind Flange Tap 1" in Center.....	BLD FLG TAP 1
Blue Print.....	B/P



Standard Abbreviations and Acronyms

Table No. 17-7 - Continued

Product or Term Identification or Description	Standard Abbreviation
Bolt Circle.....	B/C
Both Flanges Hand Tight.....	BOTH FHT
Bolt Holes.....	BH
Bolts, Alloy Steel THHN.....	BOLT THHN
Boss & Tap.....	B&T
Branch.....	BR
Bronze.....	BRZ
C	
Cement Lined.....	C/L
Center.....	CTR
Centerline.....	CTR/L
Center to End and/or Center to Flare.....	CE
Center to Face.....	CF
Center to Face-Center to PE.....	CFPE
Center to Face-Center to SKT.....	CFS
Center to SKT-Center to Face.....	CSF
Certificate.....	CERT
Chaplets.....	CHAP
Change Sheet.....	CS
Chicago District Office.....	CHI
Class.....	CL
Clockwise.....	CW
Coal Tar Epoxy.....	CTE___MILS
Coated.....	CTD
Combination.....	COMB
Common.....	COM
Compact.....	CMPT
Companion Flange.....	COMP FLG
Compression Resilient Seated.....	CRS
Concentric Reducer.....	RED
Conductive Jumper Strip.....	COND JUMPER
Copper Clip.....	CC
Counter Clockwise.....	CCW
Coupling.....	CPLG
Coupling Gland End.....	CGE
Cross.....	CROSS
Customer.....	CUST
Cut to Suit in Field.....	CTSIF
Cylinder.....	CYL
D	
Dallas District Office.....	DAL
Delivery.....	DEL
Department.....	DEPT
Description.....	DESC
Destination.....	DEST
Diameter.....	DIA



Standard Abbreviations and Acronyms

Table No. 17-7 - Continued

Product or Term Identification or Description	Standard Abbreviation
Difference.....	DIFF
Double.....	DBL
Double Cement Lined.....	DCL
Drill.....	DR
Duck Back Duck Tip.....	DT
Ductile Iron.....	DI
Ductile Iron Centrifugally Cast Pipe or Pipe Fabricated from Centrifugally Cast Pipe.....	PIPE
Ductile Iron Pipe.....	DIP
Ductile Iron Pipe Research Association.....	DIPRA
E	
Each.....	EA
Eccentric.....	ECC
Eccentric Reducer.....	ECC RED
Electric.....	ELECT
Extension.....	EXT
F	
Face & Drill.....	F&D
Face & Tap.....	F&T
Fast-Grip.....	FAST-GRIP
Fastite Conductive Gasket.....	FST PRCE GSK
Fastite Extra Deflection Socket.....	FST XDF
Fastite Joint Clamp.....	FST JT CLAMP
Fastite Lubricant.....	FST LUBE
Fastite Neoprene Gasket.....	FST NP GSK
Fastite Pipe.....	FST PIPE
Fastite Plain Rubber Gasket.....	FST GSK
Fastite Socket.....	FST
Feet.....	FT
Field Flex-Ring.....	FFR
Fitting.....	FTG
Flange(d).....	FLG or F
Flange Back Outlet.....	FLG BACK OUTLET
Flange Filler.....	FLG FILLER
Flange Flange Flange Casting, Ductile (Statically Cast).....	FFF WALL CASTING DI
Flange Flange Flange Tee.....	FLG TEE
Flange-Thrust Collar-Flange Wall Pipe, Ductile screwed-on Flanges, Welded-on Thrust Collar, Thrust Collar Centered	F-TC-F WALL PIPE
Flange Flange Pipe, Ductile Iron (Fabricated).....	FF PIPE
Flange Flange Pipe, Ductile Iron (Statically Cast).....	FF SPOOL DI
Flange Flange 90 Bend.....	FLG 90 BEND
Flange Hand Tight.....	FHT or FLG-HT
Flange Heel Outlet.....	FHO
Flange & Plain End Pipe, Ductile Iron (Fabricated).....	FPE PIPE (Designate GA)
Flange Side Outlet.....	FSO
Flare.....	FLARE
Flexible.....	FLEX
Flexible Grooved End.....	VEG (F)
Flex-Lok Ball.....	FLXBALL
Flex-Lok Pipe.....	FLEXLOK PIPE (or FLXLK)



Standard Abbreviations and Acronyms

Table No. 17-7 - Continued

Product or Term Identification or Description	Standard Abbreviation
Flex-Lok Socket.....	FLXSKT
Flex-Ring.....	FR or XR
Flex-Ring End.....	FRE or XRE
Fluoroelastomer (Fluorel, Viton)*.....	FKM
For your information.....	FYI
Freight.....	FRT
Full Length.....	FL
12x10 Flange 90 Reducing Base Bend Base Opposite 12" Flange.....	12x10 FLG 90 BASE BEND 12 CF 250 LEUP
G	
Galvanized.....	GALV
Gasket.....	GSK
Gauge Caulk Joint.....	GACJ
Gauge Dresser (___OD).....	GADR___OD
Gauge Fastite.....	GAFST
Gauge Full Length.....	GAFL
Gauge MJ.....	GAMJ
Gland.....	GLD
Gravity Microtunneling.....	GMT
Gravity Microtunneling End.....	GMTE
Gray Cast Iron.....	GI
Grooved End, Grooved End Pipe, Ductile Iron.....	VEG VEG PIPE (R or F)
H	
Hex Head Bolt Black Machine.....	HHHN BL MACH
Hex Head Hex Nut.....	HHHN
Hole.....	H (HOLE)
Hydrant.....	HYD
Hydrostatic Test.....	HYDRO/T
I	
Inside Diameter.....	ID
Invert.....	INV
Iron Body Bronze Mounted.....	IBBM
J	
Joint Material.....	JT MTL
K	
Kansas City District Office.....	KC
L	
Large End.....	LE
Laying Length.....	LL
Length.....	LGT
Less Than Truckload.....	LTL
Lever and Spring.....	L&S
Lever and Weight.....	L&W
Lining, Coating, Testing.....	LCT

*These are "brand" or "trade" names. Abbreviation is the acronym for the generic designation.



Standard Abbreviations and Acronyms

Table No. 17-7 - Continued

Product or Term Identification or Description	Standard Abbreviation
Lok-Ring Bell.....	LKR
Lok-Ring End.....	LKRE
Lok-Ring Pipe (Cut Pipe).....	LKR LKRE PIPE
Lok-Ring Pipe (Full Length).....	LKR PIPE
Long Pattern.....	LG PAT
Long Radius.....	LR
Lubricant.....	LUBE
Pound.....	LB
M	
Manufacture.....	MFG
Maximum.....	MAX
Mechanical Joint.....	MJ
Mechanical Joint Coupled Joint End.....	MJCJE
Mechanical Joint Tee.....	MJ TEE
Microtunneling.....	MT
Millimeter.....	MM
Minimum.....	MIN
Minneapolis District Office.....	MPLS
MJ & Plain End, Ductile (Cut Pipe).....	MJPE PIPE
MJ & Plain End, Ductile (Full Length).....	MJ PIPE
MJ Armor Tip.....	MJARMT
MJ Fabricated Bell.....	MJ-FAB
MJ MJ 90 Bend.....	MJ 90 BEND
Monoloy.....	MON
Scrap Metal Index.....	M/I
N	
Neoprene.....	NP
Nitrile-Butadiene (Hycar, Buna-N, Crynac, Chemigum)*.....	NBR
No Sealcoat.....	NSC
Nominal.....	NOM
Non Rising Stem.....	NRS
Not Coated.....	NOT CTD
O	
On Hand.....	OH
One Flange Hand Tight.....	1FHT
Operator.....	OPER
Original Order.....	ORIG ORDER
Orlando District Office.....	ORL
Outside Diameter.....	OD
Outside Screw and Yoke.....	OS & Y
Overall Length.....	OA

*These are "brand" or "trade" names. Abbreviation is the acronym for the generic designation.



Standard Abbreviations and Acronyms

Table No. 17-7 - Continued

Product or Term Identification or Description	Standard Abbreviation
P	
Pittsburgh District Office.....	PITT
Plain End Gauge Dresser.....	DRPE
Plain End Gauge Fastite.....	FSTPE
Plain End Gauge Fastite & Dresser.....	FTDRPE
Plain End Gauge MJ.....	MJPE
Plain End No Gauge.....	PE
Plain End Gauge Push-On.....	POPE
Plain Rubber.....	PR
Position.....	POS
Pound.....	LB
Projected On Hand.....	POH
Purchase Order/Push-On Socket.....	PO
Push-On Joint.....	POJ
Q	
Quantity.....	QTY
R	
Required.....	REQD
Retainer.....	RET
Retainer Gland.....	RET GLD
Rigid Grooved End.....	VGE (R)
Concentric Reducer.....	RED
S	
Sacramento District Office.....	SAC
Scrap Metal Index.....	M/I
Service.....	SRVC
Shipping List.....	S/L
Sleeve.....	SLV
Small End.....	SE
Socket.....	SKT
Solid Sleeve.....	SLV
Special.....	SPL
Spigots.....	SPGT
Split Sleeve.....	SPLIT SLV
Split Tapping Sleeve.....	SPLIT TAP SLV
Statically Cast Flange Flange Pipe.....	SPOOL
Statically Cast Wall Pipe.....	WALL CASTING
Studs.....	STUDS
Style.....	ST
Victaulic Shoulder End.....	VSE
T	
Tap.....	TAP
Tap 1" Mueller.....	TAP 1 MUELLER
Tap 1" NPT.....	TAP 1
Tap___At Position___	TAP___POS___
Tap For Studs.....	TFS



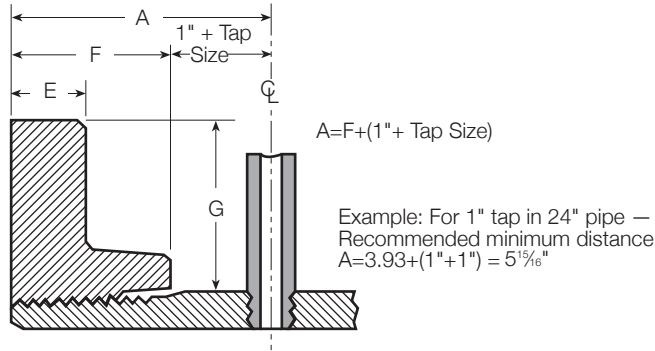
Standard Abbreviations and Acronyms

Table No. 17-7 - Continued

Product or Term Identification or Description	Standard Abbreviation
Tapping Saddle Gasket.....	TAP SADDLE GSK
Tee.....	TEE
Tee-Head Hex Nut.....	THHN
Temporary End Fabrication.....	TEFAB
Test Certificates.....	TEST CERT
Thick.....	THK
Threaded.....	THRD
Thrust Collar.....	TC
Top Dead Center.....	TDC
Transition Sleeve.....	TRAN SLV
Truck.....	TRK
U	
Uncoated.....	NOT CTD
Underwater Lubricant.....	U/W LUBE
V	
Grooved End & Grooved End Pipe.....	VGE PIPE
Victaulic.....	VICT
Victaulic Grooved End (Flexible or Rigid).....	VGE (F) or (R)
Victaulic Shoulder End Style____(41 or 44).....	VSE ST____
W	
Statically Cast Wall Pipe.....	Wall Casting
Wall Collar.....	W/C
Wall Collar (Not in Center).....	W/C____From____
Wye.....	WYE
X	
Y	
Z	



Taps in Ductile Iron Pipe*



Recommended Minimum Distance — for Small-Diameter Tap — From Face of Pipe Flange to Centerline of Tap

Table No. 17-8

Parent Pipe Size in.	Dimensions in Inches			Parent Pipe Size in.	Dimensions in Inches		
	E	F	G		E	F	G
4	.94	1.88	2.10	24	1.88	3.93	3.10
6	1.00	2.06	2.05	30	2.12	4.50	3.38
8	1.12	2.25	2.23	36	2.38	5.12	3.85
10	1.19	2.44	2.45	42	2.62	5.75	4.25
12	1.25	2.68	2.90	48	2.75	6.38	4.35
14	1.38	2.87	2.85	54	3.00	7.00	4.58
16	1.44	3.06	3.05	60	3.12	7.38	5.70
18	1.56	3.31	2.75	64	3.38	7.38	7.17
20	1.69	3.50	2.95				

* While this table provides specific rules governing the placement of individual taps relative to threaded companion flanges on flanged pipe, designers must also respect the positioning of taps relative to other pipe features (bells, wall collars, welded-on thrust collars, other taps, welded-on outlets, etc.). Contact AMERICAN for detailed information if necessary for specific applications.

Figures are based on AWWA C115 Flanges.

If closer tolerance needed, submit details of layout for recommendations.

Standard Pipe Taps—Outside Diameters

Table No. 17-9

Pipe Tap Size in.	Actual Outside Diameter in.	Pipe Tap Size in.	Actual Outside Diameter in.	Pipe Tap Size in.	Actual Outside Diameter in.
1/4	.540	1	1.315	2 1/2	2.875
3/8	.675	1 1/4	1.660	3	3.500
1/2	.840	1 1/2	1.900	-	-
3/4	1.050	2	2.375	-	-

Above data is based on direct tap into wall of pipe; see Table Nos. 17-10 and 17-11.



Taps in Ductile Iron Pipe ANSI/ASME B1.20.1

Thickness that will affect two, three and four full threads for different size taps with ANSI/ASME B1.20.1 Standard Taper Pipe Threads.

Table No. 17-10

Pipe Size in.	No. of Threads	Tap Size - in.									
		1/2	3/4	1	1 1/4	1 1/2	2	2 1/2	3	3 1/2	4
		Pipe Thickness - in.									
3	2	.18	.21	.28							
3	3	.26	.29	.37							
3	4	.33	.36	.46							
4	2	.17	.19	.26	.31						
4	3	.25	.27	.35	.40						
4	4	.32	.34	.44	.49						
6	2	.17	.18	.23	.27	.30					
6	3	.25	.26	.32	.36	.39					
6	4	.32	.33	.41	.45	.48					
8	2	.16	.17	.22	.24	.27	.33				
8	3	.24	.25	.31	.33	.36	.42				
8	4	.31	.32	.40	.42	.45	.51				
10	2	.15	.17	.21	.23	.25	.30	.44			
10	3	.23	.25	.30	.32	.34	.39	.56			
10	4	.30	.32	.39	.41	.43	.48	.69			
12	2	.15	.16	.20	.22	.24	.28	.40	.48		
12	3	.23	.24	.29	.31	.33	.37	.52	.60		
12	4	.30	.31	.38	.40	.42	.46	.65	.73		
14	2	.15	.16	.20	.22	.23	.26	.38	.45	.51	.58
14	3	.23	.24	.29	.31	.32	.35	.50	.58	.64	.70
14	4	.30	.31	.38	.40	.41	.44	.63	.70	.76	.83
16	2	.15	.16	.20	.21	.22	.25	.37	.43	.48	.54
16	3	.23	.24	.29	.30	.31	.34	.50	.56	.60	.66
16	4	.30	.31	.38	.39	.40	.43	.62	.68	.73	.79
18	2	.15	.15	.19	.21	.22	.24	.35	.41	.46	.51
18	3	.23	.23	.28	.30	.31	.33	.48	.54	.58	.64
18	4	.30	.30	.37	.39	.40	.42	.60	.66	.71	.76
20	2	.15	.15	.19	.20	.21	.23	.34	.39	.44	.49
20	3	.23	.23	.28	.29	.30	.32	.46	.52	.56	.62
20	4	.30	.30	.37	.38	.39	.41	.59	.64	.69	.74
24	2	.14	.15	.19	.20	.21	.22	.32	.37	.40	.45
24	3	.22	.23	.28	.29	.30	.31	.44	.50	.52	.58
24	4	.29	.30	.37	.38	.39	.40	.57	.62	.65	.70
30	2	.14	.15	.19	.19	.20	.21	.31	.34	.37	.41
30	3	.22	.23	.28	.28	.29	.30	.44	.46	.50	.54
30	4	.29	.30	.37	.37	.38	.39	.56	.59	.62	.66
36	2	.14	.14	.18	.19	.20	.21	.30	.33	.35	.38
36	3	.22	.22	.27	.28	.29	.30	.42	.46	.48	.50
36	4	.29	.29	.36	.37	.38	.39	.55	.58	.60	.63
42	2	.14	.14	.18	.19	.19	.20	.29	.32	.34	.36
42	3	.22	.22	.27	.28	.28	.29	.42	.44	.46	.48
42	4	.29	.29	.36	.37	.37	.38	.54	.57	.59	.61
48	2	.14	.14	.18	.18	.19	.20	.29	.31	.32	.35
48	3	.22	.22	.27	.27	.28	.29	.42	.44	.44	.48
48	4	.29	.29	.36	.36	.37	.38	.54	.56	.57	.60
54	2	.15	.15	.18	.19	.19	.20	.28	.30	.32	.34
54	3	.22	.22	.27	.27	.28	.28	.41	.43	.44	.46
54	4	.29	.29	.35	.36	.36	.37	.53	.55	.57	.59
60	2	.14	.14	.18	.19	.19	.20	.28	.30	.31	.33
60	3	.22	.22	.27	.27	.28	.28	.41	.42	.44	.46
60	4	.29	.29	.35	.36	.36	.37	.53	.55	.56	.58
64	2	.14	.14	.18	.18	.19	.20	.28	.30	.31	.33
64	3	.22	.22	.27	.27	.27	.28	.41	.42	.44	.45
64	4	.29	.29	.35	.36	.36	.37	.53	.55	.56	.50



Taps in Ductile Iron Pipe AWWA C800 Standard

Thickness that will affect two, three and four full threads for different size taps with AWWA C800 Standard Corporation Stop Threads.*

Table No. 17-11

Pipe Size in.	No. of Threads	Tap Size - in.					
		1/2	5/8	3/4	1	1 1/4	1 1/2
		Pipe Thickness - in.					
3	2	.21	.24	.25	.33		
3	3	.29	.32	.33	.41		
3	4	.36	.39	.40	.49		
4	2	.19	.22	.23	.30	.36	
4	3	.27	.30	.31	.38	.45	
4	4	.34	.37	.38	.46	.54	
6	2	.18	.20	.20	.26	.30	.35
6	3	.26	.28	.28	.34	.39	.44
6	4	.33	.35	.35	.42	.48	.53
8	2	.17	.18	.19	.24	.27	.31
8	3	.25	.26	.27	.32	.36	.40
8	4	.32	.33	.34	.40	.45	.49
10	2	.17	.17	.18	.23	.25	.28
10	3	.25	.25	.26	.31	.34	.37
10	4	.32	.32	.33	.39	.43	.46
12	2	.16	.17	.17	.22	.24	.26
12	3	.24	.25	.25	.30	.33	.35
12	4	.31	.32	.32	.38	.42	.44
14	2	.16	.17	.17	.21	.23	.25
14	3	.24	.25	.25	.29	.32	.34
14	4	.31	.32	.32	.37	.41	.43
16	2	.16	.16	.17	.21	.22	.24
16	3	.24	.24	.25	.29	.31	.33
16	4	.31	.31	.32	.37	.40	.42
18	2	.15	.16	.16	.20	.21	.23
18	3	.23	.24	.24	.28	.30	.32
18	4	.30	.31	.31	.36	.39	.41
20	2	.15	.16	.16	.20	.21	.23
20	3	.23	.24	.24	.28	.30	.32
20	4	.30	.31	.31	.36	.39	.41
24	2	.15	.15	.16	.19	.21	.22
24	3	.23	.23	.24	.27	.30	.31
24	4	.30	.30	.31	.35	.39	.40
30	2	.15	.15	.16	.19	.20	.21
30	3	.23	.23	.24	.27	.29	.30
30	4	.30	.30	.31	.35	.38	.39
36	2	.14	.15	.15	.19	.20	.20
36	3	.22	.23	.23	.27	.29	.29
36	4	.29	.30	.30	.35	.38	.38
42	2	.14	.14	.15	.18	.19	.20
42	3	.22	.22	.23	.26	.28	.29
42	4	.29	.29	.30	.34	.37	.38
48	2	.14	.14	.15	.18	.18	.19
48	3	.22	.22	.23	.26	.27	.28
48	4	.29	.29	.30	.34	.36	.37
54	2	.14	.14	.14	.17	.18	.19
54	3	.22	.22	.22	.25	.27	.28
54	4	.29	.29	.29	.34	.36	.36
60	2	.14	.14	.14	.17	.18	.19
60	3	.22	.22	.22	.25	.27	.28
60	4	.29	.29	.29	.34	.36	.36
64	2	.14	.14	.15	.17	.18	.19
64	3	.22	.22	.22	.25	.27	.28
64	4	.29	.29	.29	.34	.36	.36

*This thread is commonly known to the trade as the Mueller thread. Based on data in Appendix of AWWA C151.



Linear Expansion of Ductile Iron Pipe

The coefficient of linear expansion of ductile iron may be taken as 0.0000062 ($.62 \times 10^{-5}$) per degree Fahrenheit. The expansion or contraction in inches that will take place in a line of given length with various temperature changes is shown in the following table:

Table No. 17-12

Temp Difference °F	Length of Line in Feet				
	20'	100'	500'	1000'	5280'
	Expansion or Contraction in Inches				
5	0.007	0.04	0.19	0.37	1.96
10	0.015	0.07	0.37	0.74	3.93
20	0.030	0.15	0.74	1.49	7.86
30	0.045	0.22	1.12	2.23	11.78
40	0.060	0.30	1.49	2.98	15.71
50	0.074	0.37	1.86	3.72	19.64
60	0.089	0.45	2.23	4.46	23.57
70	0.104	0.52	2.60	5.21	27.50
80	0.119	0.60	2.98	5.95	31.43
90	0.134	0.67	3.35	6.70	35.35
100	0.149	0.74	3.72	7.44	39.28
120	0.179	0.89	4.46	8.93	47.14
150	0.223	1.12	5.58	11.16	58.92

Note: For related information, the approximate coefficient of linear expansion of other pipe and construction materials are as follows:

Concrete - $.7 \times 10^{-5}$ in./in./°F
HDPE - 10 to 12 $\times 10^{-5}$ in./in./°F
PVC - 3.1×10^{-5} in./in./°F
Steel - $.65 \times 10^{-5}$ in./in./°F



This bridge crossing illustrates design/construction advantages, including the deflection capabilities of AMERICAN Flex-Ring Joint Pipe.



Pipe Length Calculations for Offset Connections

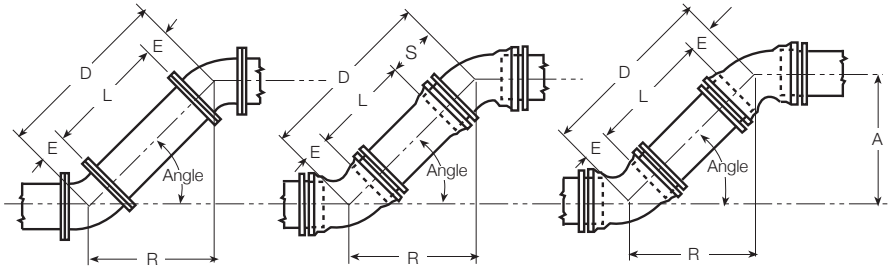


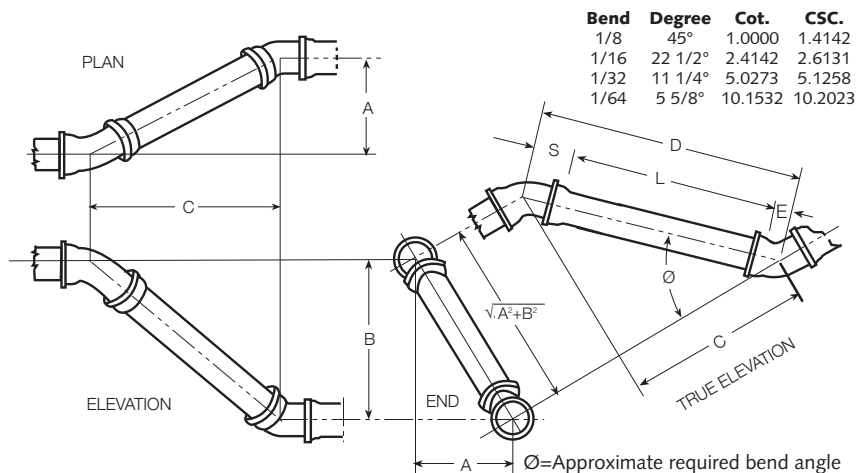
Table No. 17-13

Angle	D Equals	R Equals	L Equals	
			Flg Pipe	FST Pipe
45°	A x 1.41	A x 1.00	D - (2 x E)	D - (2 x E) or D - (E + S)
22 1/2°	A x 2.61	A x 2.41	D - (2 x E)	D - (2 x E) or D - (E + S)
11 1/4°	A x 5.13	A x 5.03	D - (2 x E)	D - (2 x E) or D - (E + S)
5 5/8°	A x 10.20	A x 10.15	D - (2 x E)	D - (2 x E) or D - (E + S)

Allowance in flange joint (usually 1/8" for gasket) and in Fastite joint (usually 1/4") should be taken into account in determination of required pipe length. Likewise, extension of restrained joints subjected to thrust load in installation and/or service should be considered as well.



Pipe Length Calculations for Inclined Diagonal Runs



Sketch explanatory of formulas for lengths of pipe for diagonal runs and angle of fittings

The following examples illustrate means of determining lengths of pipe and tie rods on inclined diagonal runs, and finding the approximate angle, \emptyset , required when the dimensions in three planes are known.

A. — When measurements in two planes are known and angle of fitting \emptyset decided upon, then

$$C = \sqrt{A^2 + B^2} \times \cot \emptyset \text{ Formula (I)}$$

$$D = \sqrt{A^2 + B^2} \times \csc \emptyset \text{ Formula (II)}$$

Example: If $A = 8'-0"$, $B = 9'-6"$, $\emptyset = 45^\circ$
Find C and length of pipe required on 12" line.

$$D = \sqrt{(8')^2 + (9.5')^2} \times \csc 45^\circ$$

$$= \sqrt{64 + 90.25} \times 1.4142$$

$$= 17.56' = 17'-6\frac{3}{4}"$$

$$C = \sqrt{(8')^2 + (9.5')^2} \times \cot 45^\circ$$

$$= \sqrt{64 + 90.25} \times 1$$

$$= 12.42 \times 1 = 12.42' = 12'-5"$$

$$L = D - (E + S)$$

$$= 17'-6\frac{3}{4}" - (5\frac{1}{2}" + 13\frac{1}{2}")$$

$$= 15'-11\frac{3}{4}"$$

B. — When measurements in three planes are known, then

$$D = \sqrt{A^2 + B^2 + C^2} \text{ Formula (III)}$$

$$\tan \emptyset = \frac{\sqrt{A^2 + B^2}}{C} \text{ Formula (IV)}$$

$$\text{Example: If } A = 5'-6", B = 3'-0",$$

$$C = 14'-9"$$

Find \emptyset and length of pipe required on 8" line.

$$D = \sqrt{(5.5')^2 + (3')^2 + (14.75')^2}$$

$$= \sqrt{30.25 + 9 + 217.56} = 16.025'$$

$$= 16'-0\frac{1}{16}"$$

$$\tan \emptyset = \frac{\sqrt{(5.5')^2 + (3')^2}}{14.75} = \frac{\sqrt{30.25 + 9}}{14.75}$$

$$= \frac{\sqrt{39.25}}{14.75} = \frac{6.265}{14.75} = 0.42474$$

$\emptyset = 23^\circ$ and a one-sixteenth bend may be used.

$$L = D - (E + S)$$

$$= 16'-0\frac{1}{16}" - (5\frac{1}{2}" + 13\frac{1}{2}")$$

$$= 14'-5\frac{5}{16}"$$

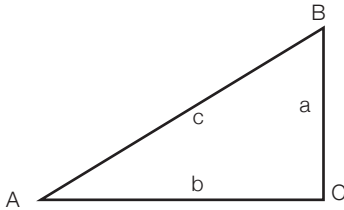
For center-to-socket dimensions of Fastite fittings see Section 7. For center-to-face dimensions of flange fittings see Section 9.

Allowance in flange joint (usually $\frac{1}{8}"$ for gasket) and in mechanical joint (usually $\frac{1}{4}"$) should be taken into account in determination of required pipe length. Likewise, extension of restrained joints subjected to axial thrust load in installation and/or service should be considered as well.



Mathematical Formulas

Solution of Right Triangle



$$a^2 + b^2 = c^2$$

$$\sin A = \frac{a}{c}$$

$$\cos A = \frac{b}{c}$$

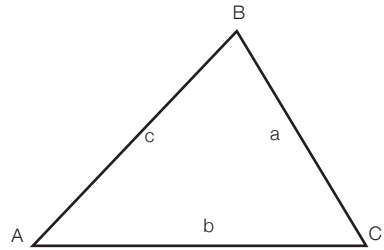
$$\tan A = \frac{a}{b}$$

$$\cot A = \frac{b}{a}$$

$$\sec A = \frac{c}{b}$$

$$\csc A = \frac{c}{a}$$

Solution of Oblique Triangle



$$A + B + C = 180^\circ$$

Law of Sines

$$\frac{a}{\sin A} = \frac{b}{\sin B} = \frac{c}{\sin C}$$

Law of Cosines:

$$c^2 = a^2 + b^2 - 2ab \cos C$$

Law of Tangents

$$\frac{a+b}{a-b} = \frac{\tan 1/2 (A+B)}{\tan 1/2 (A-B)}$$

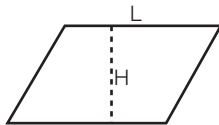
Plane Figures

A = Area

C = Circumference

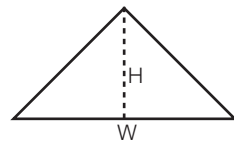
$\pi = 3.1415926536$

Parallelogram



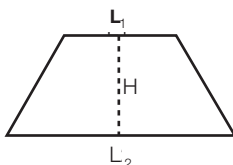
$$A = H \times L$$

Triangle



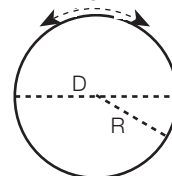
$$A = 1/2 WH$$

Trapezoid



$$A = 1/2 H(L_1 + L_2)$$

Circle
C



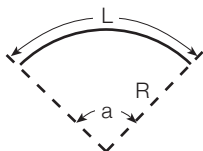
$$C = \pi D$$

$$A = 1/4 \pi D^2$$



Mathematical Formulas - Continued

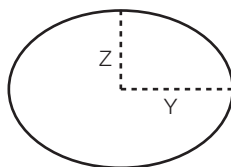
Segment of a Circle



$$A = \pi R^2 \times a \div 360$$

$$L = 2\pi R \times a \div 360$$

Ellipse



$$A = \pi ZY$$

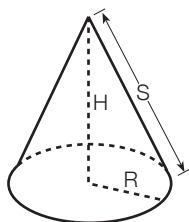
$$C = 2\pi \sqrt{1/2(Z^2 + Y^2)} \quad (\text{approximate formula})$$

Solid Figures

A = Surface Area

V = Volume

Cone

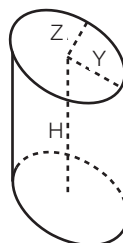


$$A = \pi R (S + R)$$

$$A_1 = \pi RS \quad (\text{Lateral Area})$$

$$V = \frac{\pi}{3} R^2 H$$

Elliptical Tank

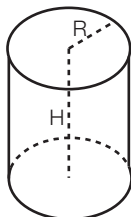


$$A = 2\pi ZY + 2\pi H \sqrt{1/2(Z^2 + Y^2)}$$

$$V = \pi ZYH$$

$$A_1 = 2\pi H \sqrt{1/2(Z^2 + Y^2)} \quad (\text{Lateral Area})$$

Cylinder

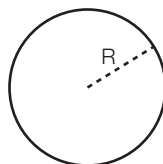


$$A = 2\pi R (H + R)$$

$$A_1 = 2\pi RH \quad (\text{Lateral Area})$$

$$V = \pi R^2 H$$

Sphere



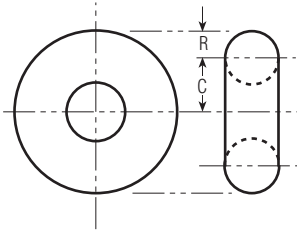
$$A = 4\pi R^2$$

$$V = \frac{4}{3} \pi R^3$$



Mathematical Formulas—Continued

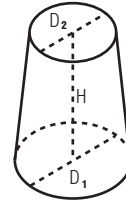
Torus



$$A = 4\pi^2 CR$$

$$V = 2\pi^2 CR^2$$

Frustum of Cone



$$V = \frac{1}{12} \pi H [(D_1^2 + D_2^2) + (D_1 \times D_2)]$$



AMERICAN's STANDARD COLOR CODES

AMERICAN routinely paints different colored marks on certain products to help differentiate them and facilitate identification.

Painted daub(s) on face of bell

Pink	Coal Tar Epoxy lining
Red	Full-length pipe cut shorter than nominal as allowed by AWWA Standards

Painted face of bell

White	Special deflection Fastite bells
-------	----------------------------------

Painted band immediately behind bell

Blue	Pipe with taps, welded-on bosses or outlets. (Blue circle also painted around the tap, boss, or outlet.)
Green	Pipe gauged full length for field cutting

Notes:

- Products to be furnished with special coatings, primers, etc., on the O.D. will not have color codings.
- Color codings are subject to change without notice. Contact AMERICAN for current codings.
- If special color codes are required to meet specific project requirements, the above colors should be avoided if at all possible.
- Other markings in addition to color codes may be painted or stencilled on products.



Decimal of an Inch and of a Foot

Table No. 17-14

Fractions of Inch or Foot		Inch Equivalents to Foot Fractions	Fractions of Inch or Foot		Inch Equivalents to Foot Fractions	Fractions of Inch or Foot		Inch Equivalents to Foot Fractions	Fractions of Inch or Foot		Inch Equivalents to Foot Fractions
1/64	.0052	1/16	17/64	.2552	3/16	33/64	.5052	6/16	49/64	.7552	9/16
	.0104	1/8		.2604	3/8		.5104	6/8		.7604	9/8
	.015625	3/16		.265625	3/16		.515625	6/16		.765625	9/16
	.0208	1/4		.2708	3/4		.5208	6/4		.7708	9/4
	.0260	5/16		.2760	3/16		.5260	6/16		.7760	9/16
1/32	.03125	3/8	9/32	.28125	3/8	17/32	.53125	6/8	25/32	.78125	9/8
	.0365	7/16		.2865	3/16		.5365	6/16		.7865	9/16
	.0417	1/2		.2917	3/2		.5417	6/2		.7917	9/2
	.046875	9/16		.296875	3/16		.546875	6/16		.796875	9/16
3/64	.0521	5/8	19/64	.3021	3/8	35/64	.5521	6/8	51/64	.796875	9/16
	.0573	11/16		.3073	31/16		.5573	61/16		.8073	91/16
	.0625	3/4		.3125	3/4		.5625	6/4		.8125	9/4
	.0677	13/16		.3177	31/16		.5677	61/16		.8177	91/16
	.0729	7/8		.3229	3/8		.5729	6/8		.8229	9/8
5/64	.078125	15/16	21/64	.328125	31/16	37/64	.578125	61/16	53/64	.828125	91/16
3/32	.0833	1	11/32	.3333	4	19/32	.5833	7	27/32	.8333	10
	.0885	11/16		.3385	41/16		.5885	71/16		.8385	101/16
	.09375	11/8		.34375	41/8		.59375	71/8		.84375	101/8
	.0990	17/16		.3490	41/16		.5990	71/16		.8490	101/16
	.1042	11/4		.3542	41/4		.6042	71/4		.8542	101/4
7/64	.109375	17/16	23/64	.359375	41/16	39/64	.609375	71/16	55/64	.859375	101/16
	.1146	13/8		.3646	41/8		.6146	71/8		.8646	101/8
	.1198	17/16		.3698	41/16		.6198	71/16		.8698	101/16
	.1250	11/2		.3750	41/2		.6250	71/2		.8750	101/2
1/8	.1302	17/16	3/8	.3802	41/16	5/8	.6302	71/16	7/8	.8802	101/16
	.1354	11/8		.3854	41/8		.6354	71/8		.8854	101/8
	.140625	111/16		.390625	411/16		.640625	711/16		.890625	1011/16
	.1458	11/4		.3958	41/4		.6458	71/4		.8958	101/4
	.1510	113/16		.4010	413/16		.6510	713/16		.9010	1013/16
5/32	.15625	17/8	13/32	.40625	41/8	21/32	.65625	71/8	29/32	.90625	101/8
11/64	.1615	115/16	27/64	.4115	415/16	43/64	.6615	715/16	59/64	.9115	1015/16
	.1667	2		.4167	5		.6667	8		.9167	11
	.171875	27/16		.421875	51/16		.671875	81/16		.921875	111/16
	.1771	27/8		.4271	51/8		.6771	81/8		.9271	111/8
	.1823	27/16		.4323	51/16		.6823	81/16		.9323	111/16
3/16	.1875	21/4	7/16	.4375	51/4	11/16	.6875	81/4	15/16	.9375	111/4
	.1927	21/16		.4427	51/16		.6927	81/16		.9427	111/16
	.1979	23/8		.4479	53/8		.6979	83/8		.9479	113/8
	.203125	27/16		.453125	51/16		.703125	81/16		.953125	111/16
13/64	.2083	21/2	29/64	.4583	51/2	45/64	.7083	81/2	61/64	.9583	111/2
	.2135	21/8		.4635	51/8		.7135	81/8		.9635	111/8
	.21875	23/8		.46875	53/8		.71875	83/8		.96875	113/8
	.2240	211/16		.4740	511/16		.7240	811/16		.9740	1111/16
	.2292	23/4		.4792	53/4		.7292	83/4		.9792	113/4
15/64	.234375	213/16	31/64	.484375	513/16	47/64	.734375	813/16	63/64	.984375	1113/16
1/4	.2396	21/4	1/2	.4896	51/4	3/4	.7396	81/4	1	.9896	111/4
	.2448	215/16		.4948	51/16		.7448	81/16		.9948	111/16
	.2500	3		.5000	6		.7500	9		1.0000	12



Equivalents and Conversion Factors English

The word gallon, used in any conversion factor, designates the U.S. gallon. Likewise, the word ton designates a short ton, 2,000 pounds.

The figures 10^{-1} , 10^{-2} , 10^{-3} , etc., denote 0.1, 0.01, 0.001, etc., respectively.

The figures 10^1 , 10^2 , 10^3 , etc., denote 10, 100, 1000, etc., respectively.

"Parts Per Million," (designated as p.p.m.), is always by weight. As used in the sanitary field, p.p.m. represents the number of pounds of dry solids contained in one million pounds of water. In this field, one part per million may be expressed as 8.345 pounds of dry solids to one million U.S. gallons of water.

Conversion factors are based on water at a temperature of 4°C unless otherwise noted.

Table No. 17-15

Multiply	By	To Obtain
Acres.....	43,560	Square feet
Acre-feet.....	43,560	Cubic feet
Acre-feet.....	325,851	Gallons
Atmospheres.....	29.92	Inches of mercury
Atmospheres.....	33.90	Feet of water
Atmospheres.....	14.70	Lbs/sq. inch
Barrels cement.....	376	Pounds-cement
Bags or sacks-cement.....	94	Pounds-cement
B.T.U.....	778.17	Foot-lbs
B.T.U.....	3.9301 $\times 10^{-4}$	Horsepower-hrs.
B.T.U.....	2.9306 $\times 10^{-4}$	Kilowatt-hrs.
B.T.U./min.	12.970	Foot-lbs/sec.
B.T.U./min.	0.023581	Horsepower
B.T.U./min.	0.017584	Kilowatts
Cubic feet.....	7.48052	Gallons
Cubic feet.....	1728	Cubic inches
Cubic feet.....	0.03704	Cubic yards
Cubic feet/second.....	0.646317	Million gals./day
Cubic feet/second.....	448.831	Gallons/min.
Cubic inches.....	5.787 $\times 10^{-4}$	Cubic feet
Cubic inches.....	4.329 $\times 10^{-3}$	Gallons
Cubic yards.....	27	Cubic feet
Cubic yards.....	46,656	Cubic inches
Cubic yards.....	202.0	Gallons
Drams.....	27.34375	Grains
Drams.....	0.0625	Ounces
Fathoms.....	6	Feet
Feet of water.....	0.8826	Inches of mercury
Feet of water (at 62°F).....	0.4330	Lbs/sq. inch
Feet of water.....	62.427	Lbs/sq. feet
Feet of water.....	0.02950	Atmospheres
Foot-pounds.....	1.285 $\times 10^{-3}$	B.T.U.
Foot-pounds.....	5.0505 $\times 10^{-7}$	Horsepower-hrs.
Foot-pounds.....	3.766 $\times 10^{-7}$	Kilowatt-hrs.
Foot-pounds/min.	3.0303 $\times 10^{-5}$	Horsepower
Foot-pounds/min.	2.260 $\times 10^{-5}$	Kilowatts
Gallons.....	0.13368	Cubic feet
Gallons.....	231	Cubic inches
Gallons, Imperial.....	1.20095	U.S. gallons
Gallons, U.S.....	0.83267	Imperial gallons
Gallons water (at 4°C).....	8.3453	Pounds of water
Gallons water (at 62°F).....	8.3355	Pounds of water
Gallons/min.....	2.228 $\times 10^{-3}$	Cubic feet/sec.
Gallons/min.....	8.0208	Cu.ft./hr.
Grains (troy).....	1	Grains (avoir.)



Equivalents and Conversion Factors English

Table No. 17-15 - Continued

Multiply	By	To Obtain
Grains	0.06480	Grams
Grains/U.S. gal.	17.119	Parts/million
Grains/U.S. gal.	142.86	Lbs./million gal.
Grains/Imp. gal.	14.254	Parts/million
Hectares	2.471	Acres
Horsepower	42.407	B.T.U./min.
Horsepower	550	Foot-lbs./sec.
Horsepower	0.7457	Kilowatts
Horsepower (boiler)	33,520	B.T.U./hr.
Horsepower (boiler)	9.824	Kilowatts
Inches	1.0	Mils
Inches of mercury	1.133	Feet of water
Inches of mercury	0.4912	Lbs./sq. inch
Inches of mercury	0.03342	Atmospheres
Inches of water	0.07355	Inches of mercury
Inches of water	0.03613	Lbs./sq. inch
Kilowatts	56.87	B.T.U./min.
Kilowatts	737.5	Foot-lbs./sec.
Kilowatts	1.341	Horsepower
Miles	5280	Feet
Miles/min88	Feet/sec.
Million gals./day	1.54723	Cubic ft./sec.
Miner's inches	1.5	Cubic ft./min.
Ounces	437.5	Grains
Ounces	0.9115	Ounces (troy)
Ounces (fluid)	1.805	Cubic inches
Parts/million	8.345	Lbs./million gal.
Parts/million	0.058415	Grains/U.S. gal.
Parts/million	0.07016	Grains/Imp. gal.
Pounds	16	Ounces
Pounds	7000	Grains
Pounds	1.21528	Pounds (troy)
Pounds of water	0.01602	Cubic feet
Pounds of water	27.68	Cubic inches
Pounds of water	0.1198	Gallons
Pounds/cubic foot	5.787	Lbs./cubic inch
Pounds/sq. foot	0.01602	Feet of water
Pounds/sq. inch	2.307	Feet of water
Pounds/sq. inch	2.0358	Inches of mercury
Pounds/sq. inch	0.06803	Atmospheres
Quires	25	Sheets
Reams	500	Sheets
Square feet	144	Square inches
Square feet	2.296	Acres
Square miles640	Acres
Square miles	2.788	Square feet
Square yards9	Square feet
Square yards	2.066	Acres
Tons (long)	2240	Pounds
Tons (long)	1.12	Tons (short)
Tons (short)	2000	Pounds
Tons of water/24 hrs.	0.16639	Gallons/min.
Watts	0.05686	B.T.U./min.
Watts	0.7375	Foot-pounds/sec.
Watts	1.341	Horsepower

Conversion Factors are based on water at a temperature of 4°C unless otherwise noted.



Equivalents and Conversion Factors Metric

Table No. 17-16

Multiply	By	To Obtain
Acres.....	4047	Square meters
Acre-feet.....	1233.5	Cubic meters
Atmosphere.....	1.01325	Bars
Atmospheres.....	76.0	Centimeters of mercury
Bars.....	0.98692	Atmosphere
Bars.....	1.02	$\times 10^4$ Kgs./sq. meter
Bars.....	14.50777	Pounds/sq. in.
Bars.....	10.20	Meters of water
B.T.U.	0.2520	Kilogram calories
B.T.U.	107.6	Kilogram meters
Centimeters.....	0.3937	Inches
Centimeters of mercury.....	0.01316	Atmospheres
Centimeters of mercury.....	0.4461	Feet of water
Centimeters of mercury.....	27.85	Pounds/sq. ft.
Centimeters of mercury.....	0.1934	Pounds/sq. in.
Centimeters/second.....	1.969	Feet/minute
Centimeters/second.....	0.03281	Feet/sec.
Centimeters/second.....	0.6	Meters/min.
Cubic centimeters.....	3.531	$\times 10^{-5}$ Cubic feet
Cubic centimeters.....	6.102	$\times 10^{-2}$ Cubic inches
Cubic centimeters.....	2.642	$\times 10^{-4}$ Gallons
Cubic centimeters.....	10^{-6}	Cubic meters
Cubic centimeters.....	10^{-3}	Liters
Cubic feet.....	28.32	Liters
Cubic feet.....	2.832	$\times 10^4$ Cubic centimeters
Cubic feet.....	0.02832	Cubic meters
Cubic inches.....	16.39	Cubic centimeters
Cubic inches.....	1.639	$\times 10^{-5}$ Cubic meters
Cubic inches.....	1.639	$\times 10^{-2}$ Liters
Cubic meters.....	35.31	Cubic feet
Cubic meters.....	1.308	Cubic yards
Cubic meters.....	264.2	Gallons
Cubic meters.....	10^3	Liters
Cubic yards.....	0.7646	Cubic meters
Cubic yards.....	764.6	Liters
Dram.....	1.771845	Grams
Feet.....	30.48	Centimeters
Feet.....	0.3048	Meters
Feet of water.....	304.8	Kgs./sq. meter
Feet/sec./sec.....	0.3048	Meters/sec./sec.
Feet/sec.....	30.48	Centimeters/sec.
Feet/sec.....	18.29	Meters/minute
Foot-pounds.....	0.1383	Kilogram-meters
Gallons.....	3.785	Cubic centimeters
Gallons.....	3.785	$\times 10^{-3}$ Cubic meters
Gallons.....	3.785	Liters
Gallons/minute.....	0.06308	Liters/second
Gallons/minute.....	6.308	$\times 10^{-5}$ Cubic meters/sec.
Grams.....	15.432	Grains
Grams.....	0.03527	Ounces
Grams.....	980.7	Dynes
Grams/sq. centimeter.....	0.9808	$\times 10^{-3}$ Bars
Grams/liter.....	58.410	Grains/gal.
Grams/liter.....	8.344	Pounds/1000 gallons
Grams/liter.....	1000	Parts/million
Grams/cubic centimeter.....	1.000	$\times 10^3$ Kilograms/cubic meter
Inches.....	2.540	Centimeters
Inches.....	25.4	Millimeters (mm)
Inches of mercury.....	345.3	Kgs./sq. meter



Equivalents and Conversion Factors Metric

Table No. 17-16—Continued

Multiply	By	To Obtain
Inches/second.....	2.540 x 10 ⁻²	Meters/second
Inches/sec./sec.	2.540 x 10 ⁻²	Meters/sec./sec.
Kilograms.....	2.2046.....	Pounds
Kilogram-calories/minute.....	51.47.....	Foot-pounds/sec.
Kilogram-calories/minute.....	0.06979.....	Kilowatts
Kilogram-calories/minute.....	0.09359.....	Horsepower
Kilograms/sq. meter.....	1.422 x 10 ⁻³	Pounds/sq. inch
Kilometers.....	3281.....	Feet
Kilometers.....	0.6214.....	Miles
Kilometers/hour.....	0.9113.....	Feet/sec.
Kilometers/hour.....	27.78.....	Centimeters/sec.
Kilowatts.....	14.33.....	Kg.-calories/min.
Liters.....	0.2642.....	Gallons
Liters.....	61.02.....	Cubic inches
Liters.....	0.03531.....	Cubic feet
Meters.....	3.281.....	Feet
Meters.....	39.37.....	Inches
Meters.....	1.094.....	Yards
Meters of water.....	0.09803.....	Bars
Meters of water.....	1.422.....	Pounds/sq. inch
Miles.....	1.609.....	Kilometers
Miles/minute.....	1.609.....	Kilometers/minute
Milligrams/liter.....	1.....	Parts/million
Millimeters.....	0.03937.....	Inches
Newton/Cubic meter.....	6.365 x 10 ⁻³	Pounds/cubic foot
Newtons/sq. millimeter.....	145.045.....	Pounds/sq.inch
Ounces.....	28.3495.....	Grams
Ounces (fluid).....	29.58.....	Cubic centimeters
Pounds.....	453.5924.....	Grams
Pounds.....	4.448.....	Newtons
Pounds/cubic foot.....	0.01602.....	Grams/cubic centimeter
Pounds/cubic foot.....	16.02.....	Kilograms/cubic meter
Pounds/foot.....	1.488.....	Kilograms/meter
Pounds/inch.....	178.6.....	Grams/centimeter
Pounds/sq. inch.....	0.06893.....	Bars
Pounds/sq. foot.....	4.883.....	Kilograms/sq.meter
Pounds/sq. inch.....	2.30947.....	Feet of Water (at 62°F)
Pounds/sq. inch.....	703.1.....	Kilograms/sq.meter
Pounds/sq. inch.....	6.895.....	Kilopascals (kPa)
Pounds/sq. inch.....	6.895 x 10 ⁻³	Megapascals (MPa)
Square centimeters.....	0.1550.....	Square inches
Square centimeters.....	1.076 x 10 ⁻³	Square feet
Square feet.....	0.09290.....	Square meters
Square feet/second.....	9.290 x 10 ⁻²	Square meters/sec.
Square inches.....	6.452.....	Square centimeters
Square kilometers.....	247.1.....	Acres
Square kilometers.....	1.076 x 10 ⁷	Square feet
Square kilometers.....	1.196 x 10 ⁶	Square yards
Square meters.....	10.76.....	Square feet
Square meters.....	1.196.....	Square yards
Square miles.....	2.590.....	Square kilometers
Square yards.....	0.8361.....	Square meters
Tons (long).....	1016.....	Kilograms
Tons (metric).....	10 ³	Kilograms
Tons (metric).....	2205.....	Pounds
Tons (short).....	907.185.....	Kilograms
Watts.....	0.01433.....	Kilogram-calories/minute
Yards.....	0.9144.....	Meters

Conversion factors are based on water at a temperature of 4°C unless otherwise noted.

[illegible]